



Exceptional service in the national interest



Comparative Fluid-Structure Interaction Analysis of Solid and Hollow Core Blades in Full-Scale Reference Model 1 Tidal Turbine

Dongyoung Kim¹ and Budi Gunawan¹

¹*Water Power Technologies, Sandia National Laboratories*

UMERC+OREC Conference 2025, Corvallis, OR, Aug. 12–14

August 12, 2025

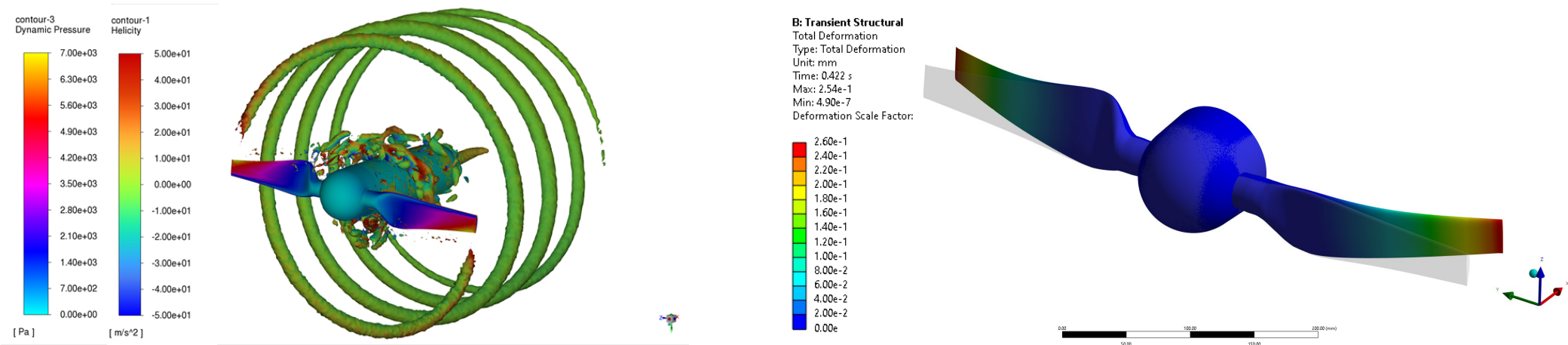


Sandia National Laboratories is a multimission laboratory managed and operated by National Technology and Engineering Solutions of Sandia, LLC, a wholly owned subsidiary of Honeywell International Inc., for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-NA0003525

SAND2025-09829C

INTRODUCTION

- ✓ Tidal turbines must maintain performance and durability under diverse and demanding loading conditions.
- ✓ The complex interaction between turbulent water flow and blade structure induces significant stress, deflection, and fatigue, making accurate prediction of device reliability and Levelized Cost of Energy (LCOE) a critical challenge.
- ✓ Traditional one-way simulations, which separate fluid and structural analysis, often fail to capture the full picture.
- ✓ This study utilizes a two-way coupled Fluid-Structure Interaction (FSI) model, providing time-accurate solutions for the loading and performance of a deforming rotor, which is crucial for realistic performance prediction.
- ✓ By focusing on the full-scale RM1 tidal turbine, we explore these dynamics to enhance the design and efficiency of future marine energy systems.



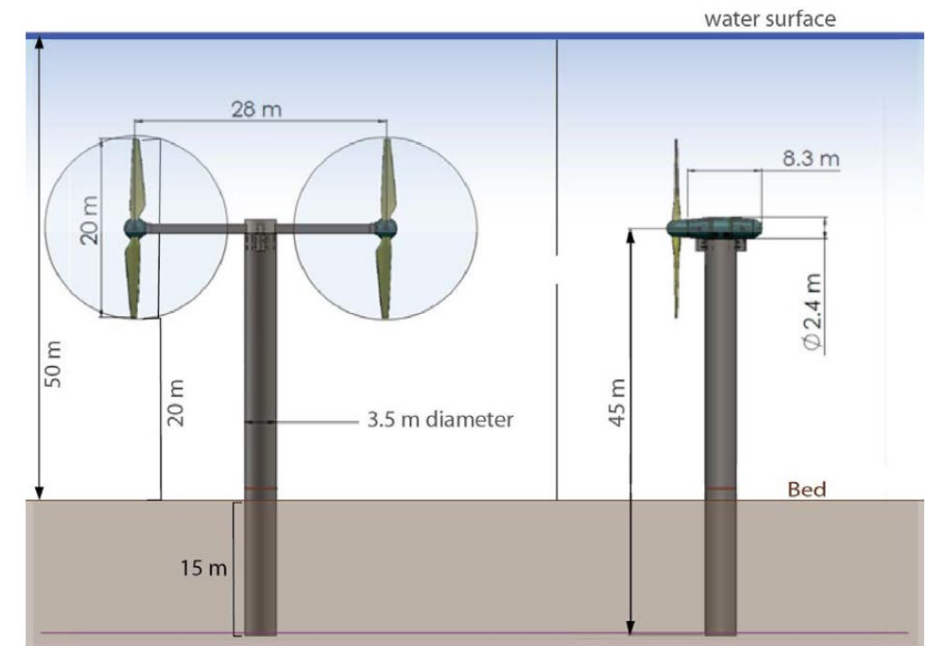
Q-criterion iso-surface colored by helicity and pressure contour on the turbine surface (left) and magnitude of total deformation of the rotor (right)

OBJECTIVES AND TARGET GEOMETRY MODEL

- ✓ Develop and verify a high-fidelity, two-way coupled FSI model for the full-scale RM1 tidal turbine.
- ✓ Compare the turbine's hydrodynamic performance (power, thrust) and structural response (deformation, stress) using both one-way and two-way FSI simulations to quantify the impact of fluid-structure coupling.
- ✓ Analyze and contrast the performance of solid vs. hollow core blade geometries to inform structural design and optimization.
- ✓ Demonstrate the necessity of two-way FSI for achieving accurate structural integrity assessments required for future LCOE analysis.

Reference Model 1 (RM1)

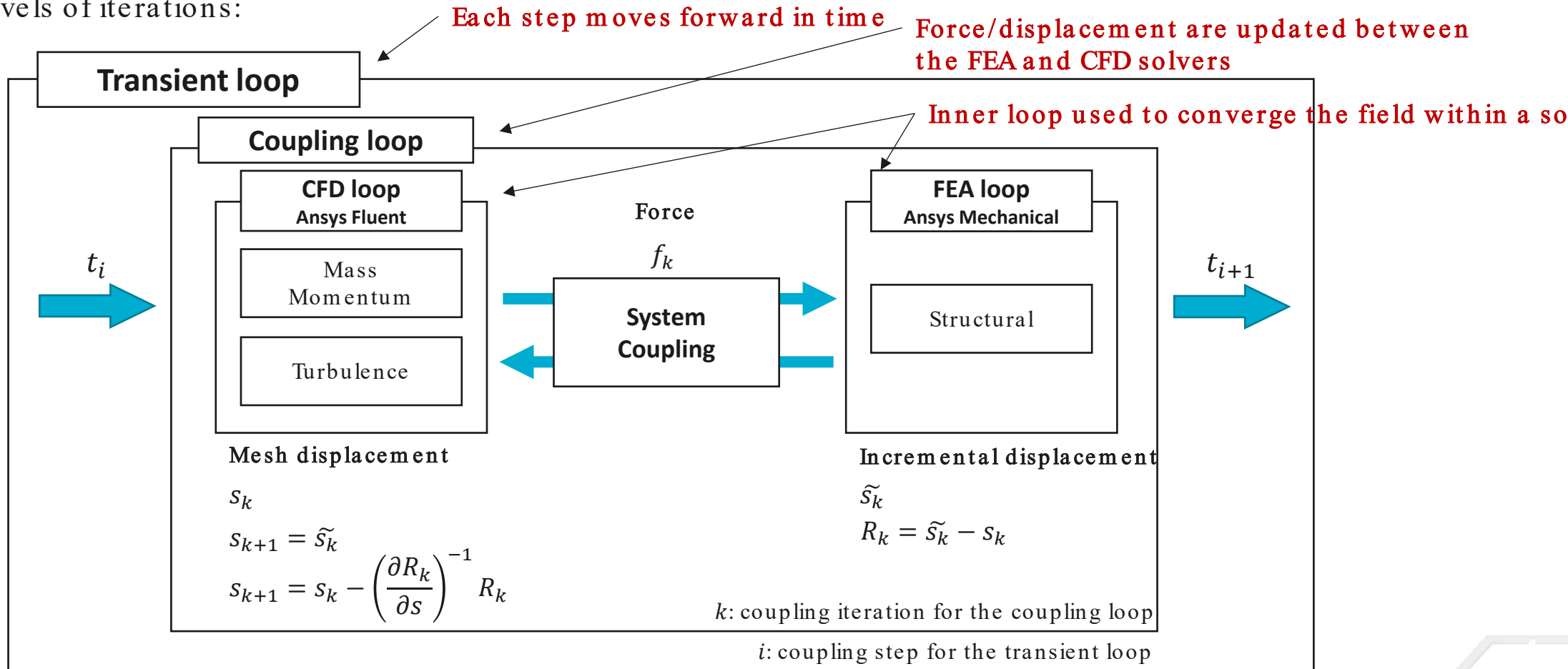
- Rotor diameter: 20m
- Rated current speed: 2.0 m/s
- Blade construction: The FSI model analyze three distinct blade configurations:
 - Solid core metal blade
 - Hollow core metal blade
 - Hollow core composite blade



TWO-WAY COUPLED FSI MODEL

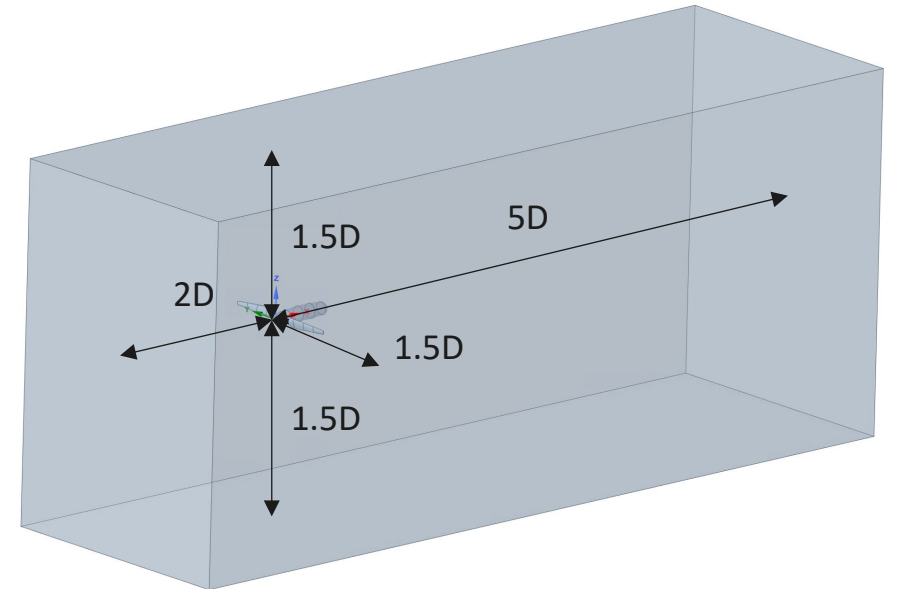
Two-way iteratively implicit approach (strong coupling)

- Iterate within each time step to obtain an implicit solution
- Three levels of iterations:



CFD MODELING

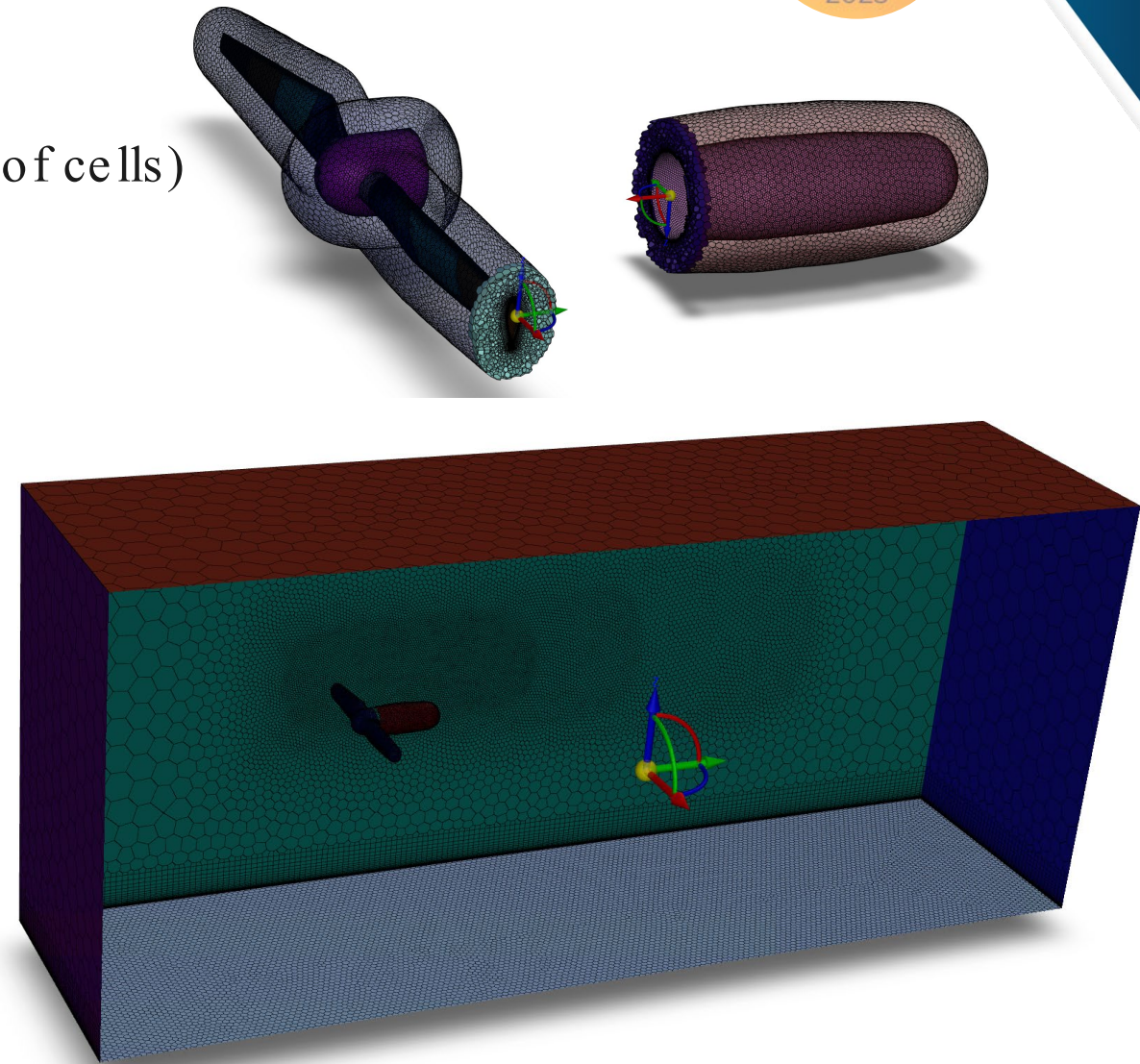
- Computational domain and boundary conditions
 - One rotor only
 - Cut off 1% of chord length for mesh quality
 - Blockage effect is ignored
 - $(0, 0, 0,)$ at the nose of the rotor
 - Inlet: 2 m/s uniform flow
 - Outlet: zero gauge pressure
 - Symmetry: top and sides
 - No-slip wall: rotor, nacelle, and bottom



CFD MODELING



- Computational Mesh (Medium grid)
 - Polyhedral mesh with overset multi-blocks (# of cells)
 - Rotor: 5.88M
 - Nacelle: 1.02M
 - Bkg w/ refinement: 2.67M
 - Total: 9.57M
 - Prism layers on the rotor and nacelle wall
 - Target $y^+ = 30 - 50$



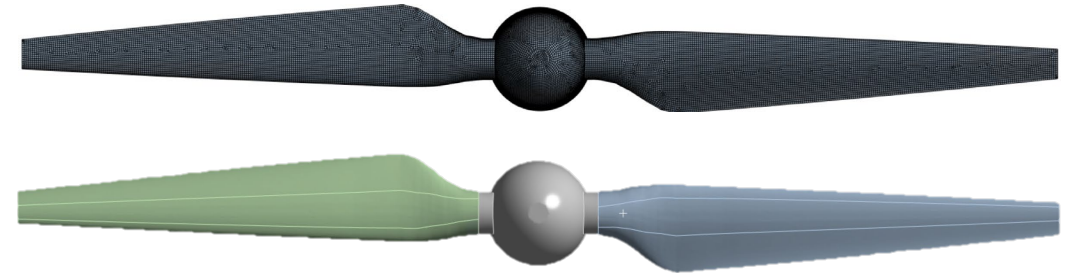
- Mathematical Model and Numerical Scheme

- Viscous model:
 - Realizable $k - \epsilon$ model with wall function
- Pressure-velocity coupling:
 - Pressure-based coupled solver
- Spatial discretization:
 - Pressure: second order
 - Momentum: second order upwind
 - Turbulence model: second order upwind
- Temporal discretization:
 - Transient formulation: first order implicit

- FEA Setup

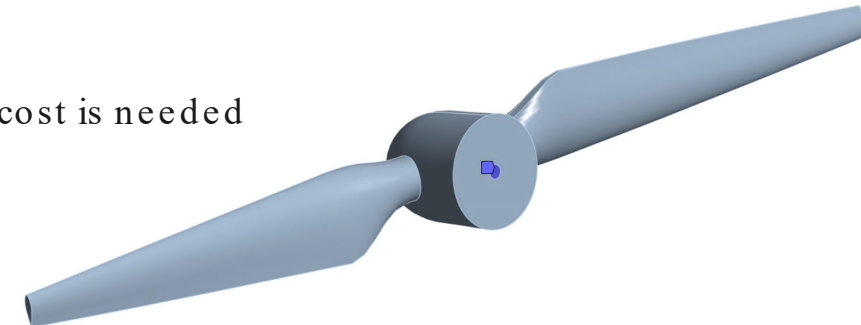
- Geometry and mesh

- Rotor only
 - Hexahedral mesh with quadratic element order
 - Metal blades are modeled as a solid and hollow blade made from aluminum alloy 6061 T6
 - Hub is modeled as a solid for hollow metal and composite blade rotors

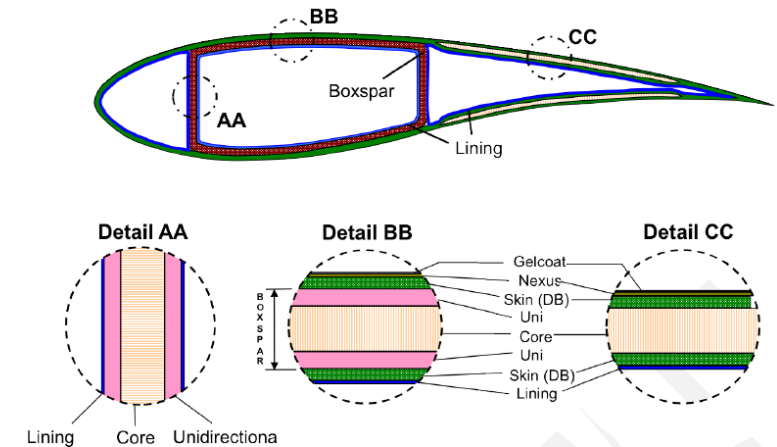
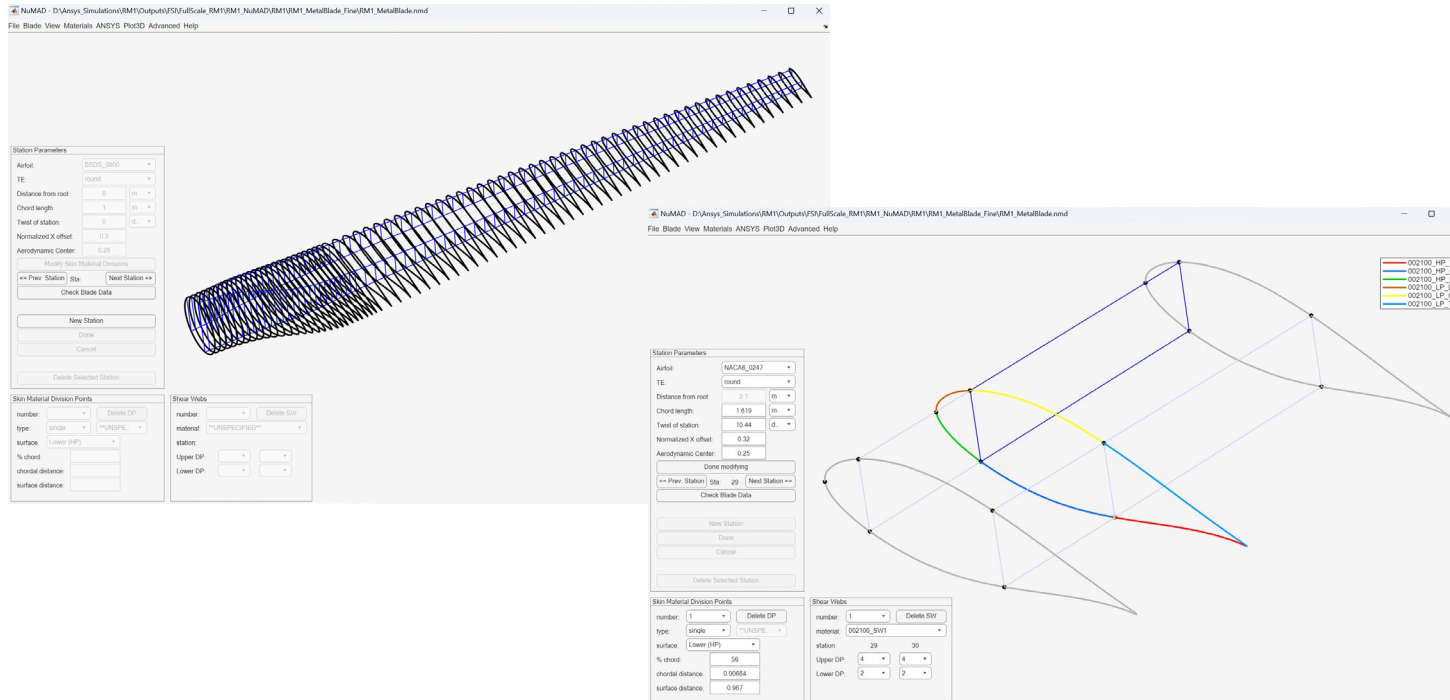


- Boundary conditions

- Assigned angular velocity corresponding to the turbine rotating speed
 - Fixed support at the rotor hub center
 - Without modeling actual rotation of rotor in FEA side, lower computation cost is needed



- Modeled hollow metal and composite blades with shear webs leveraging NuMAD
 - The blade is divided into spanwise 74 stations.
 - Each station includes 5 division points, defining 8 sections for assigning material layups.
 - Material stacking sequences and geometry definitions are referenced from Lawson et al. (2012).
 - Shear webs are positioned at 12.8% and 56.0% of the local chord length.



Material	tply (mm)	ρ (Kg/m ³)	E (Pa)	G (Pa)	σ (Pa)	σ (Pa)
Gelcoat	0.381	1664	--	--	--	--
Nexus	0.51	1830	--	--	--	--
Double-bias	0.53	1830	1.03E+10	8.00E+09	151	-174
Lining	0.53	1830	1.03E+10	8.00E+09	151	-174
Unidirectional	0.53	1860	3.70E+10	4.10E+09	986	-746
Core	3.125	128.1	--	--	--	--

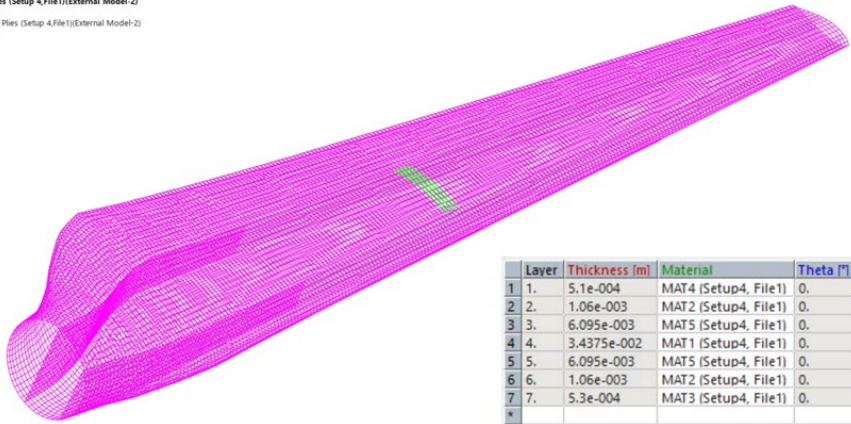
3D model of the RM1 turbine blade with shear webs generated using NuMAD (top), and skin material segmentation at station 29 (bottom)

Representative structural layup of composite laminates at a typical blade section (adapted from Lawson et al., 2012)

FEA MODELING

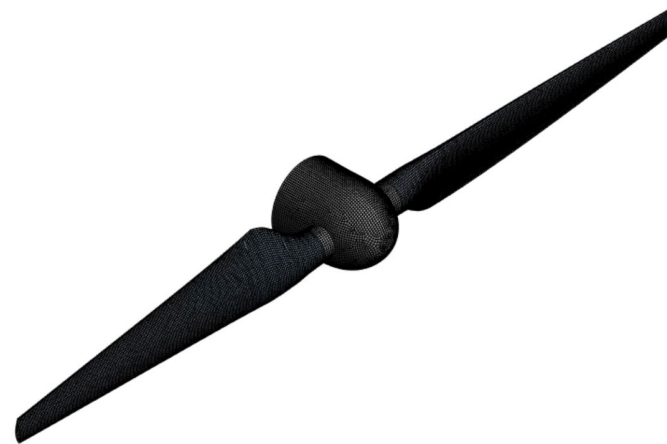
- Converted NuMAD blade model for import into ANSYS Mechanical
 - Blade shell geometry and composite ply information successfully imported into the structural FEA environment
 - Rotor hub and blade root modeled as solid parts
 - Hub–blade connection modeled as bonded (no separation or sliding)

Ply1 (Setup 4, File1)(External Model-2)
Ply2 (Setup 4, File1)(External Model-2)



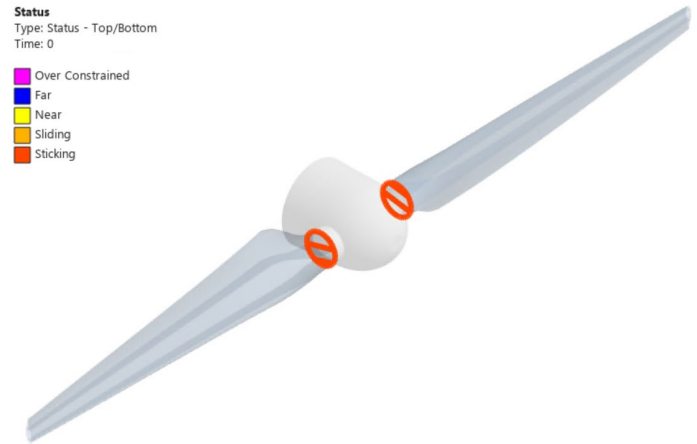
Layer	Thickness [m]	Material	Theta [°]
1	5.1e-004	MAT4 (Setup4, File1)	0.
2	1.06e-003	MAT2 (Setup4, File1)	0.
3	6.095e-003	MAT5 (Setup4, File1)	0.
4	3.4375e-002	MAT1 (Setup4, File1)	0.
5	6.095e-003	MAT5 (Setup4, File1)	0.
6	1.06e-003	MAT2 (Setup4, File1)	0.
7	5.3e-004	MAT3 (Setup4, File1)	0.

Imported composite layup configuration in ANSYS Mechanical and material stack details at a section



Status
Type: Status - Top/Bottom
Time: 0

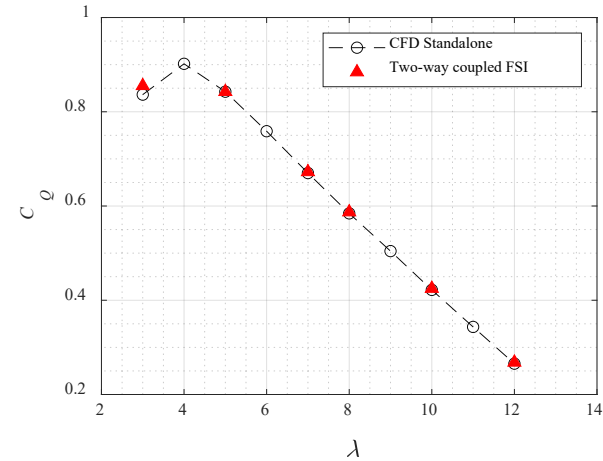
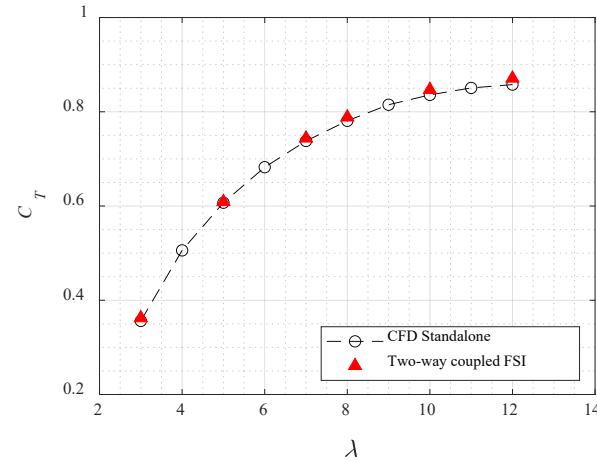
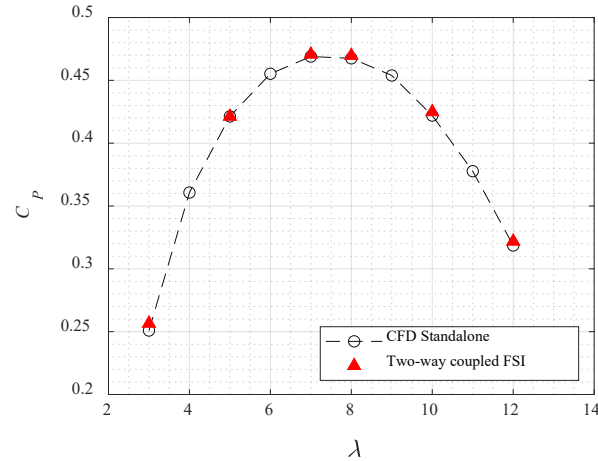
Over Constrained
Far
Near
Sliding
Sticking



FEA mesh of full-scale hollow RM1 rotor blade (left) and contact surface boundary condition (right)

SIMULATION RESULTS – SOLID METAL BLADE

CFD vs Two-way coupled FSI



TSR	C_p			C_T			C_Q		
	CFD	FSI	Diff.	CFD	FSI	Diff.	CFD	FSI	Diff.
3.0	0.251	0.257	2.23 %	0.356	0.362	1.75 %	0.836	0.855	2.23 %
5.0	0.421	0.421	-0.01 %	0.607	0.609	0.33 %	0.843	0.843	-0.01 %
7.0	0.469	0.471	0.37 %	0.738	0.744	0.78 %	0.670	0.672	0.37 %
8.0	0.468	0.470	0.46 %	0.781	0.788	0.98 %	0.584	0.587	0.46 %
10.0	0.422	0.425	0.68 %	0.836	0.847	1.32 %	0.422	0.425	0.68 %
12.0	0.319	0.322	1.02 %	0.858	0.871	1.55 %	0.265	0.268	1.02 %

The low rotational speed introduces numerical errors in modeling the buffer layer of the boundary layer over the rotor due to the use of wall functions for computational efficiency.

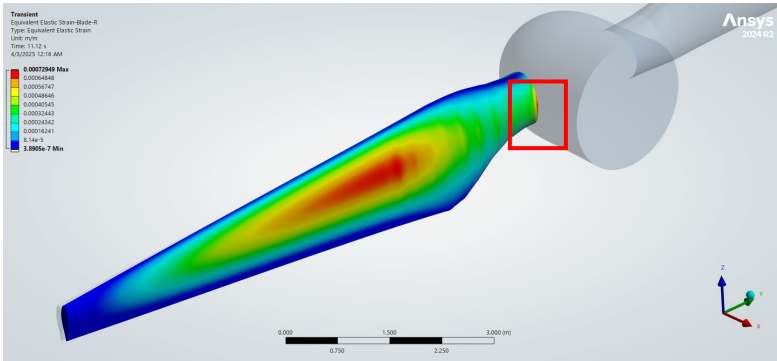
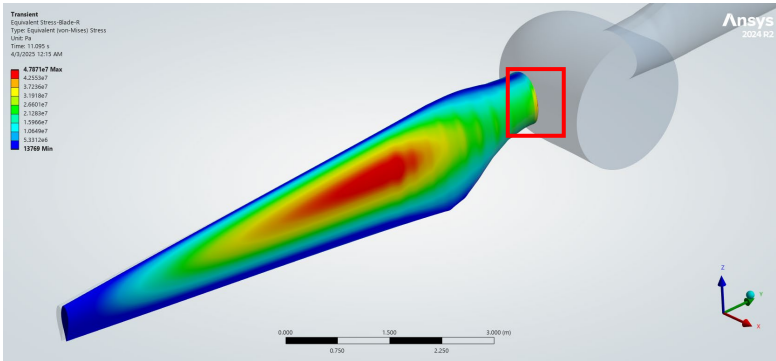
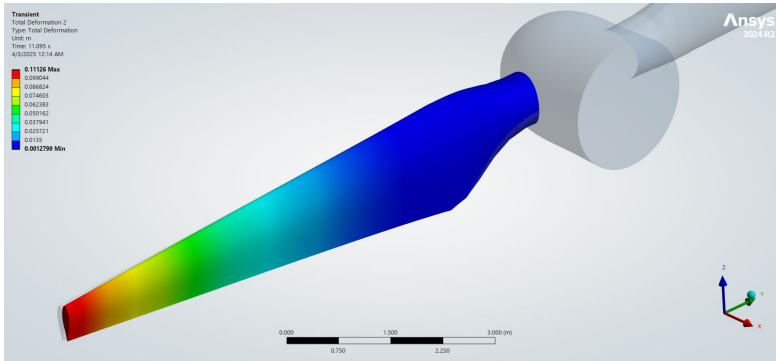
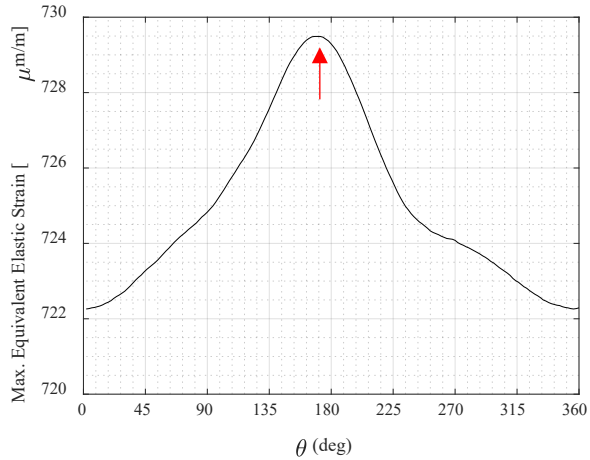
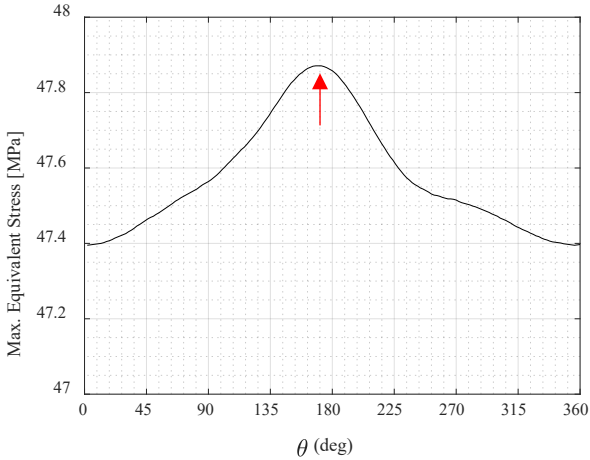
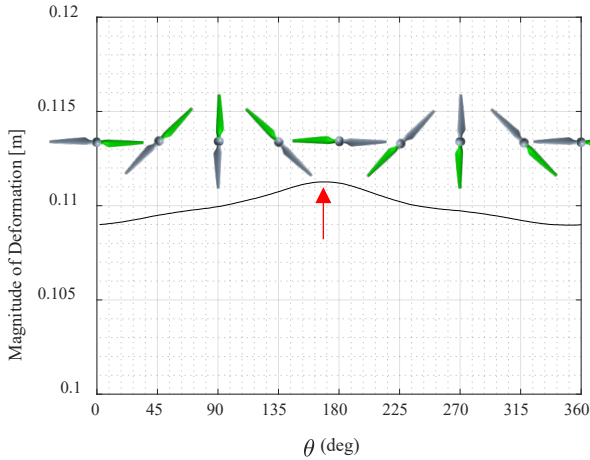
Power, thrust, and torque coefficient estimated from CFD stand alone and two-way coupled FSI simulations

SIMULATION RESULTS – SOLID METAL BLADE



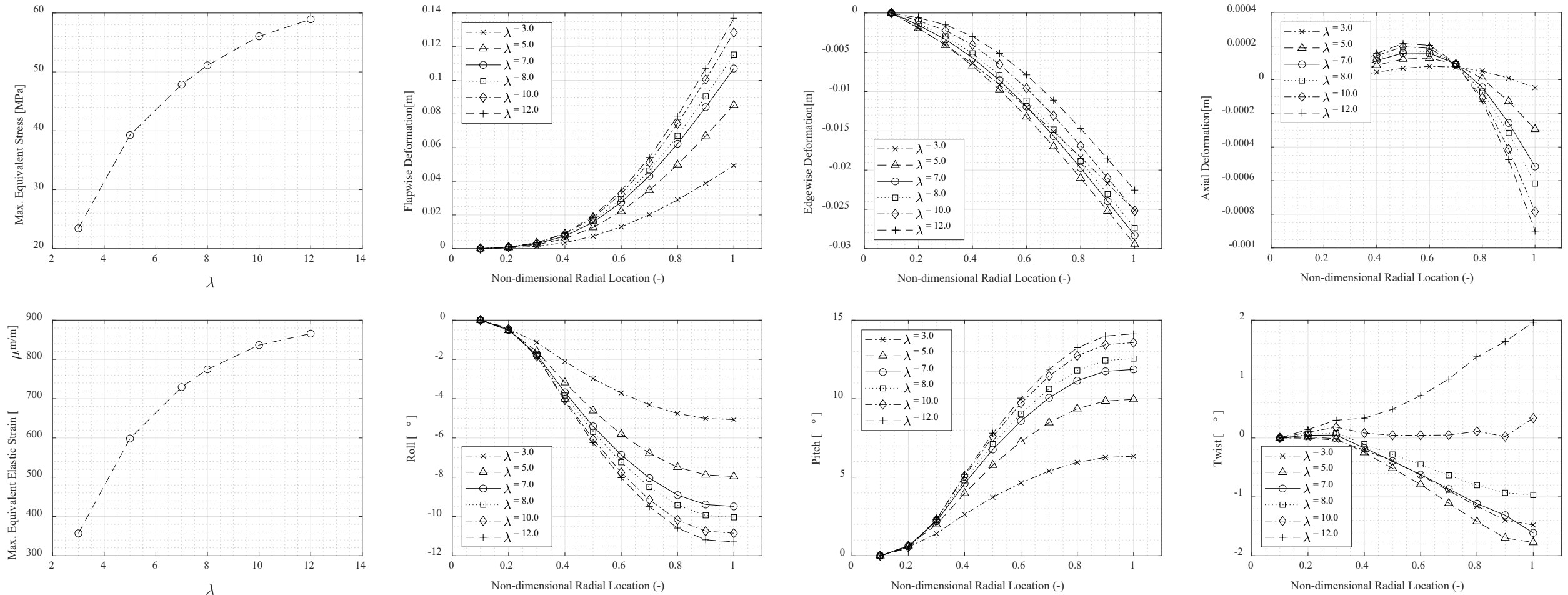
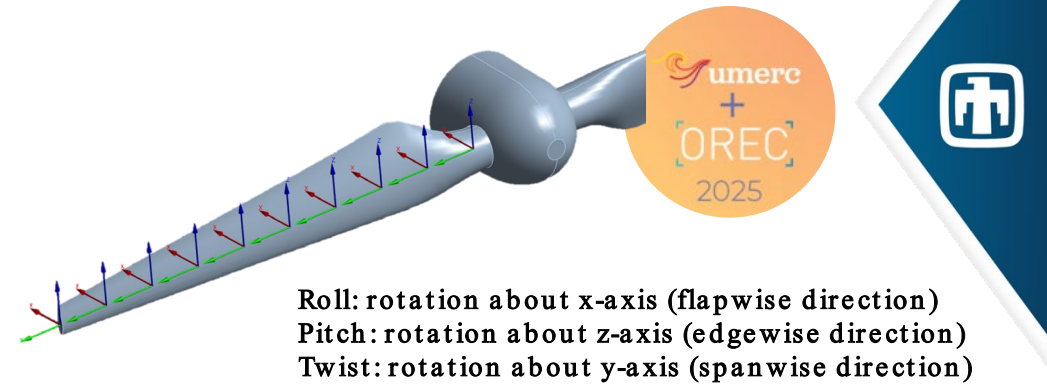
Structural response @ TSR = 7.0 (BOP)

@ BOP (TSR = 7.0)	Max. Magnitude of total deformation (m)	Max. Equivalent Stress (MPa)	Max. Equivalent Elastic Strain ($\mu\text{m/m}$)
One-way FSI	0.1016	45.72	674.79
Two-way FSI	0.1113	47.87	730.49
Diff.	5.91 %	4.50 %	7.64 %



SIMULATION RESULTS – SOLID METAL BLADE

Maximum structural response at various TSRs

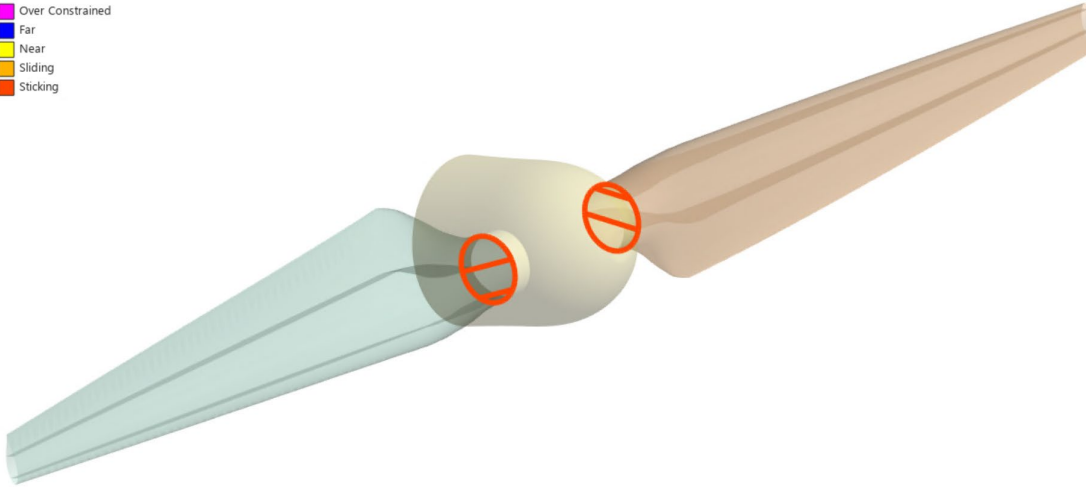


HOLLOW METAL BLADE MODEL

Full Model

- Hollow metal blade + Hub
- Direct connection between solid hub and shell blade geometries
- Predicts the stress concentration at the blade-hub interface

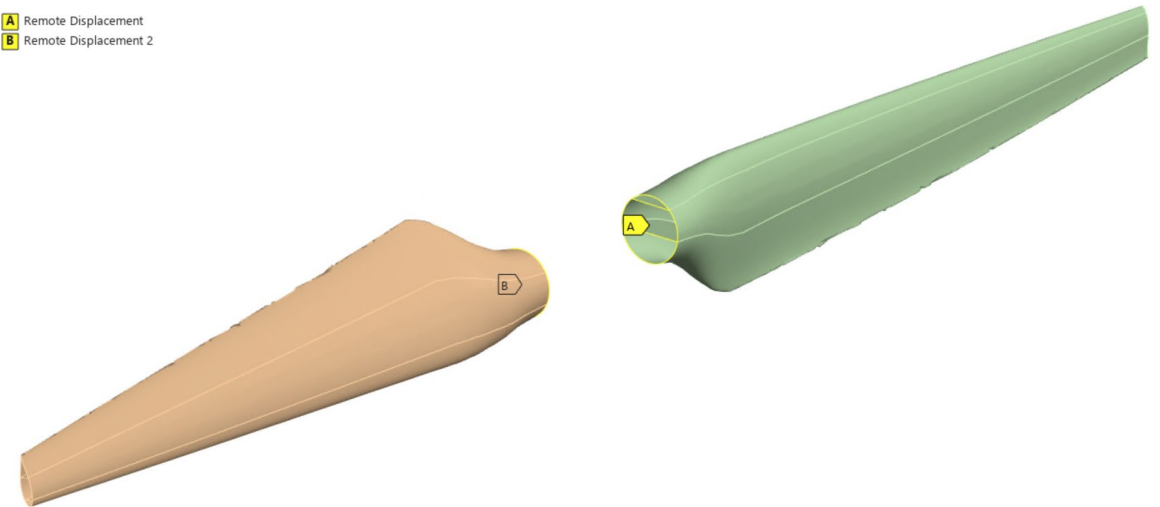
Over Constrained
Far
Near
Sliding
Sticking



Simplified Model

- Hollow metal blade only
- Remote displacement at blade root
- Cannot capture the local peak stress at the connection

A Remote Displacement
B Remote Displacement 2



SIMULATION RESULTS –METAL BLADE 1-WAY FSI



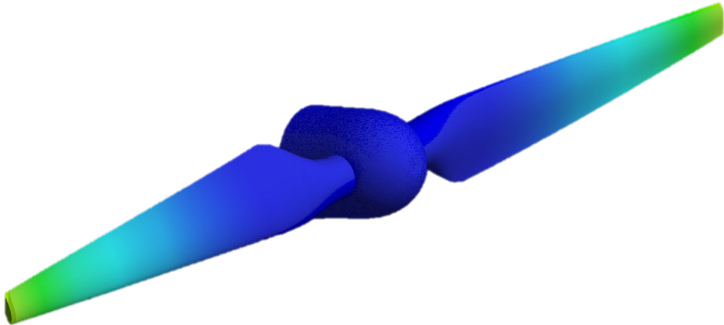
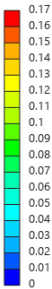
Structural response @ TSR = 7.0 (BOP)

Full Solid Blade Model

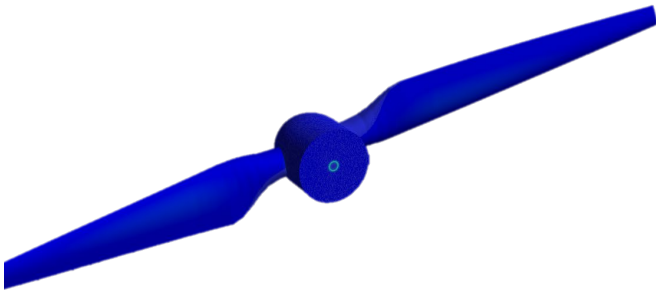
Full Hollow Blade Model

Hollow Blade Only

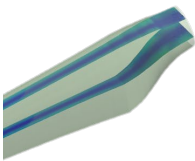
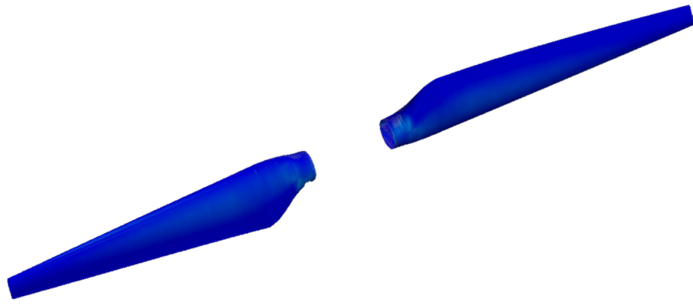
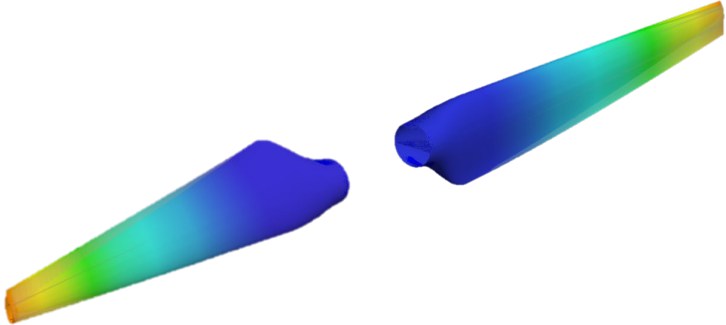
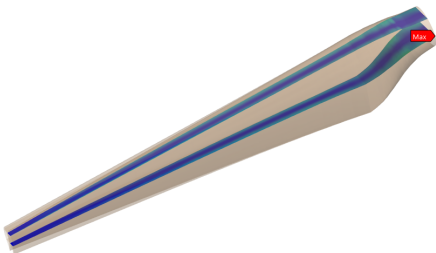
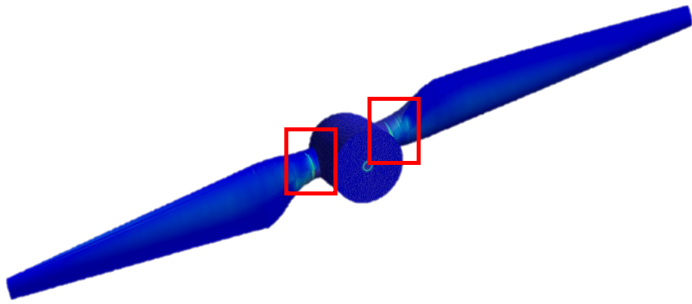
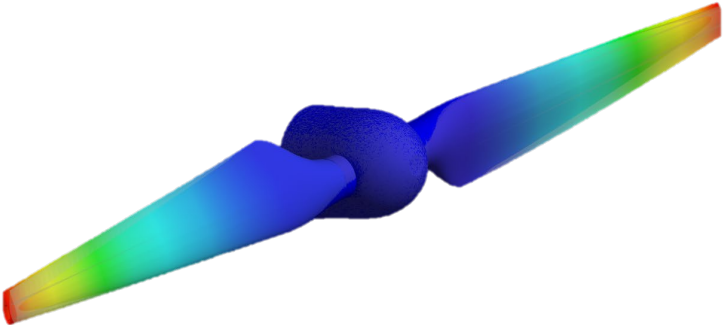
Type: Total Deformation
Unit: m
Time: 1 s
Max: 0.10469
Min: 0



Type: Equivalent (von-Mises) Stress
Unit: Pa
Time: 1 s
Max: 2.98e8
Min: 1730.2



Type: Equivalent (von-Mises) Stress - Top/Bottom - Layer 0
Unit: Pa
Time: 1 s
Max: 1.8491e
Min: 6.3193e



CONCLUSIONS



- High-Fidelity FSI Model:
 - Successfully developed and verified a two-way coupled FSI model on the solid metal blade, proving its necessity for accurate structural assessment over one-way analysis.
- Hollow Blade Workflow:
 - Established a workflow for modeling complex hollow blades by integrating NuMAD with ANSYS Mechanical.
- Identified Key Challenge:
 - Analysis of the hollow metal blade revealed a critical stress concentration at the solid-to-shell transition near the hub.
- Diagnosed Composite Instability:
 - The large stiffness difference between the hub and the highly flexible composite blade was identified as the source of numerical non-convergence.

FUTURE WORK



- Resolve Stress Concentration:
 - Modify the solid root geometry in CAD to create a gradual, tapered stiffness transition, mimicking composite ply drop-offs.
- Extend FSI Analysis:
 - Apply the validated two-way FSI simulation to the optimized hollow metal and composite blade models.
- Compare All Designs:
 - Conduct a final performance and structural reliability comparison of the solid, hollow metal, and composite blades.
- Inform LCOE:
 - Use the final, comprehensive dataset to inform future design optimization and Levelized Cost of Energy (LCOE) analysis.