

# Cable-Based Actuation for Small-Scale Model Testing of a Floating Offshore Structure

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Anisha Sharma<sup>1,2</sup>

Coauthored by Matthew Hall<sup>2</sup>, Hannah Ross<sup>2</sup>, Kathryn Johnson<sup>1,2</sup>, Will Wiley<sup>2</sup>, Senu Sirnivas<sup>2</sup>

<sup>1</sup>Colorado School of Mines, <sup>2</sup>National Renewable Energy Lab (NREL)

# Floating Offshore Structure Design

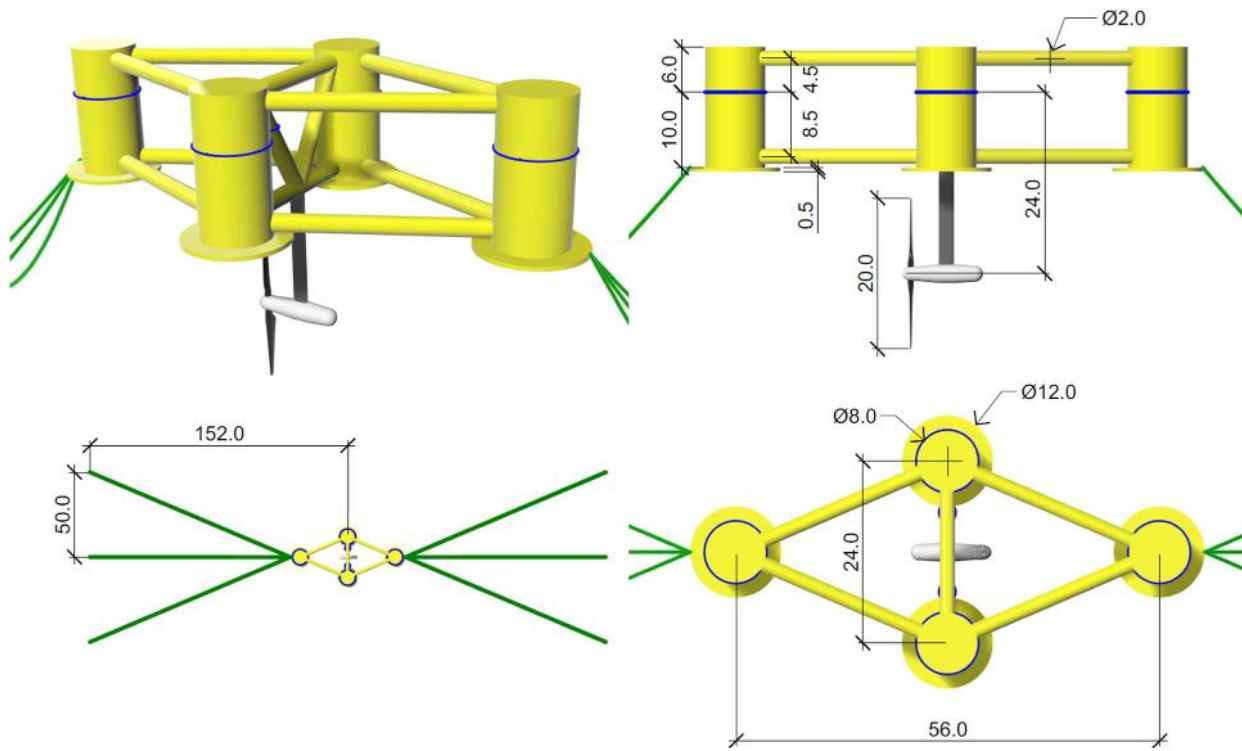


Fig 1. Reference Model 1 (RM1), a floating marine energy turbine<sup>1</sup>

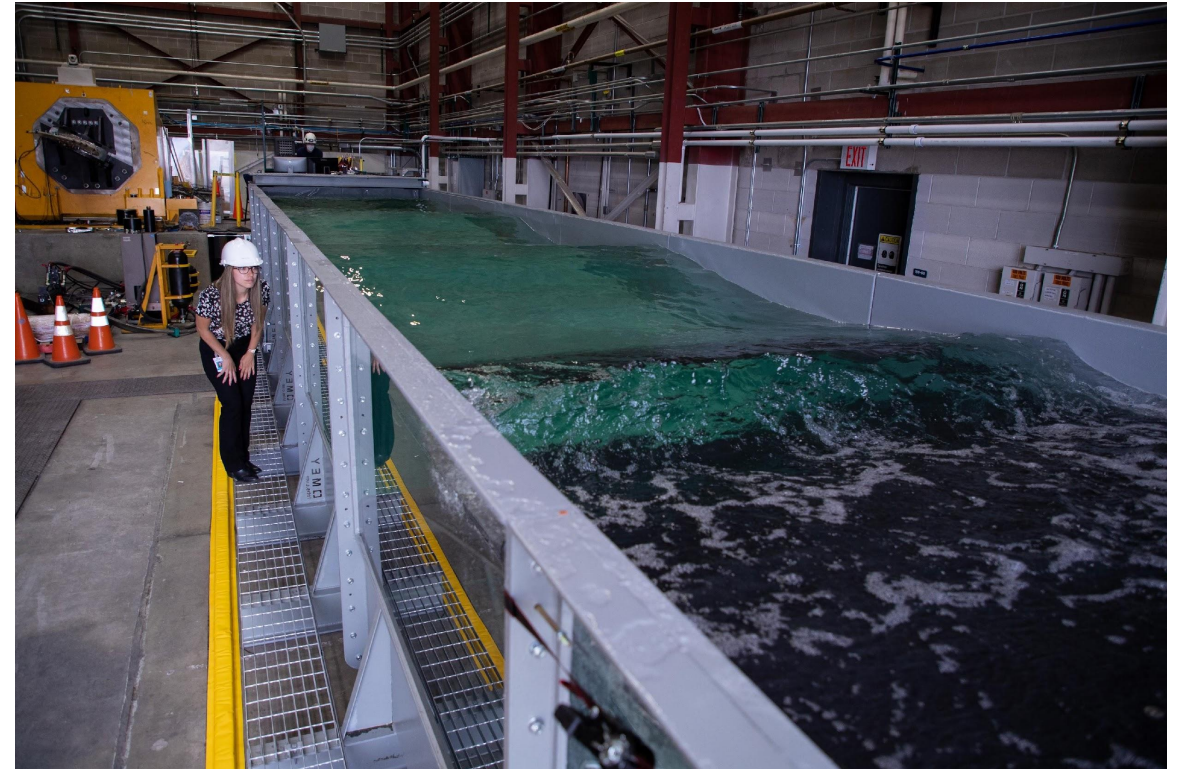


Fig 2. NREL Flatirons wave basin

<sup>1</sup>Wiley et al. (2023), <sup>2</sup>NREL Wave Tank Team

# Small-Scale Testing Challenges

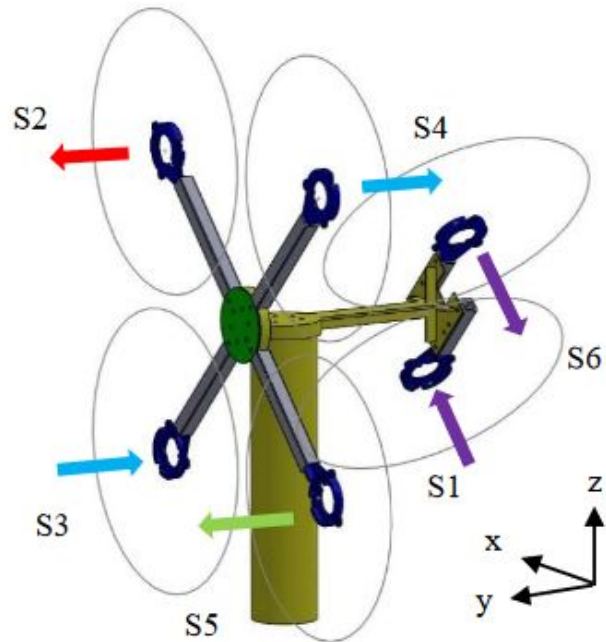


Reynolds-Froude Scaling  
Incompatibilities



Physical Limitations in  
Test Facilities

# Emulation of Aerodynamic Forces Using a Fan Based Actuation System



MPD used in this study to emulate the aerodynamic loads of the NREL 5 MW reference turbine at 1:37 scale

Fig 3. Multi-Propellor Device (MPD) developed to simulate aerodynamic loads using fans<sup>3</sup>

<sup>3</sup>Otter et al. (2020)

# Emulation of Forces Using Cable Based Actuation

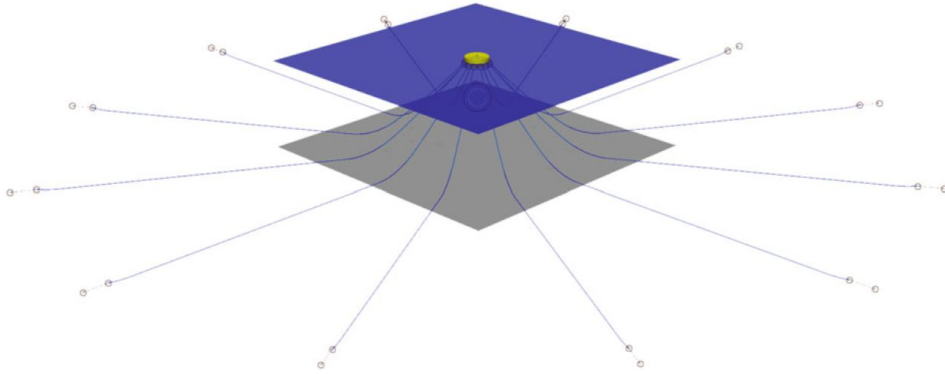


Fig 4. Representation of the emulated system<sup>4</sup>

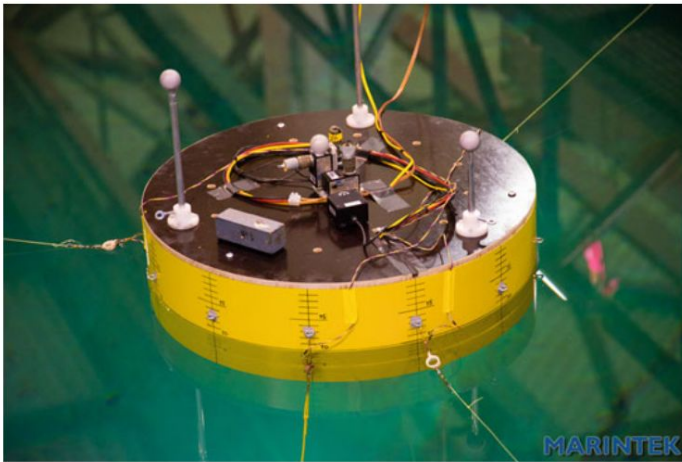
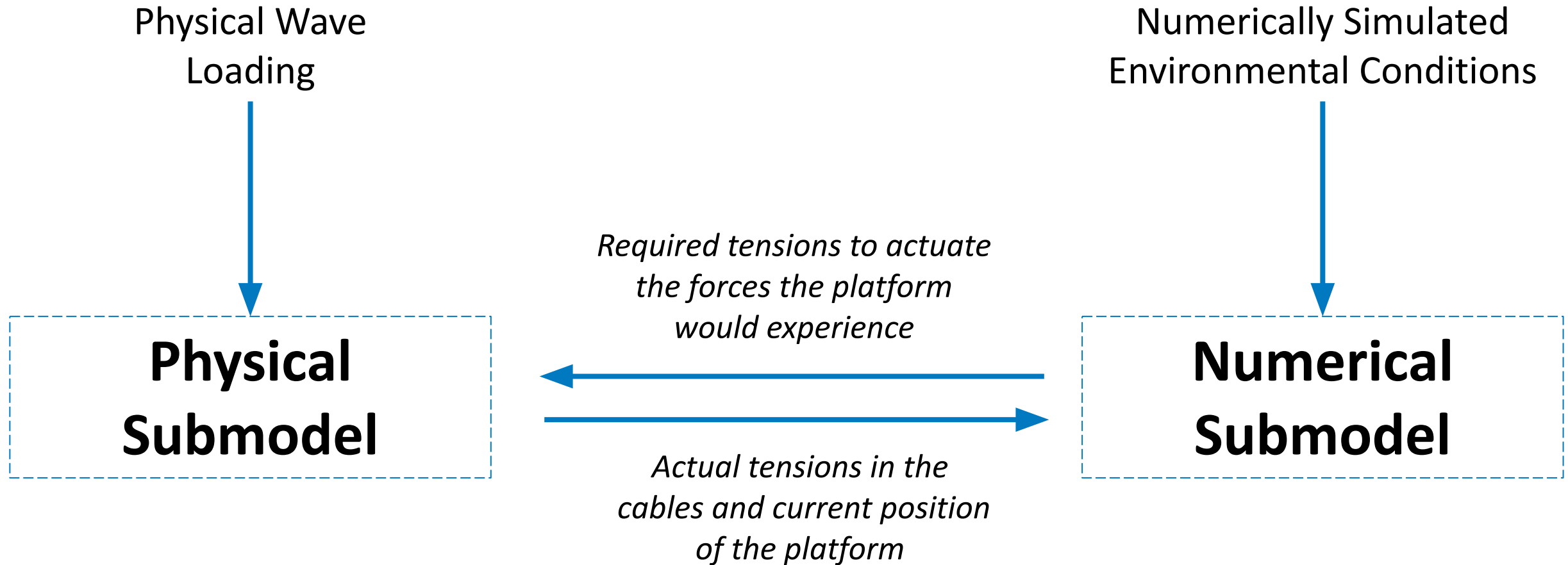


Fig 5. Model of floater in basin<sup>4</sup>

Three actuation lines (visible in Fig 5) connected to the floater applied the calculated load from a numerical simulation

<sup>4</sup>Vilsen et al. (2017)

# Hybrid Submodeling Strategy



# Hybrid Submodeling Strategy

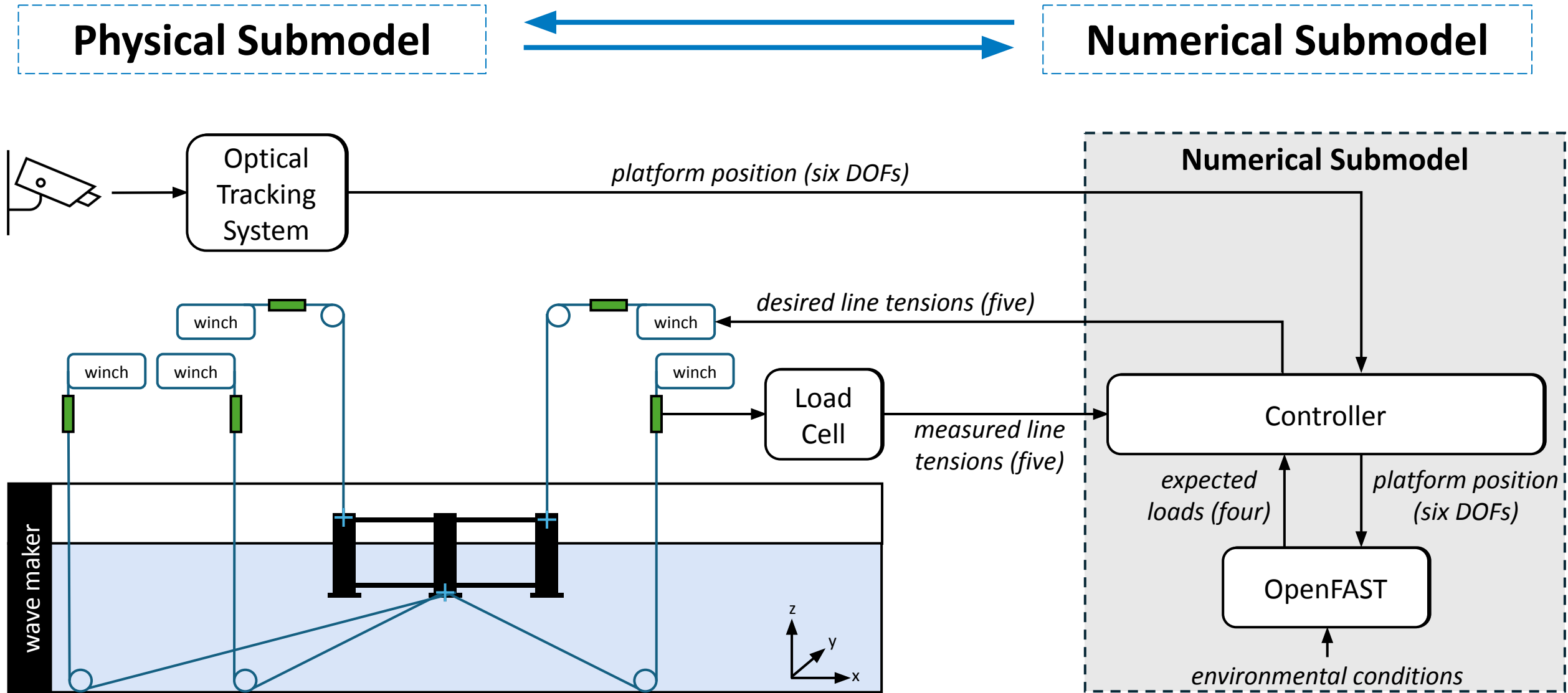


Fig 6. Overview of substructuring strategy



# Cable Tension Allocation

$$\mathbf{w} = -\mathbf{A}^T \boldsymbol{\tau}$$

*Wrench Matrix of Expected Forces = Structure Matrix from Cable Geometry x Cable Tensions*



# Cable Tension Allocation

$$\mathbf{w} = -\mathbf{A}^T \boldsymbol{\tau}$$

**Wrench Vector of Forces and Moment to be applied**

(from numerical model)

$$\mathbf{w} = \begin{Bmatrix} F_x \\ F_y \\ F_z \\ M_y \end{Bmatrix}$$

where

$F_x$  is force along x-axis

$F_y$  is force along y-axis

$F_z$  is force along z-axis

$M_y$  is moment around y-axis

**Structure Matrix**

(from geometry of cables)

$$\mathbf{A}^T = \begin{Bmatrix} \mathbf{u}_1 & \dots & \mathbf{u}_5 \\ \mathbf{b}_1 \times \mathbf{u}_1 & \dots & \mathbf{b}_5 \times \mathbf{u}_5 \end{Bmatrix}$$

where

$\mathbf{u}_m$  is the unitary cable direction vector

$\mathbf{b}_m$  is the cable point of attachment vector

**Tension Vector of Cables**

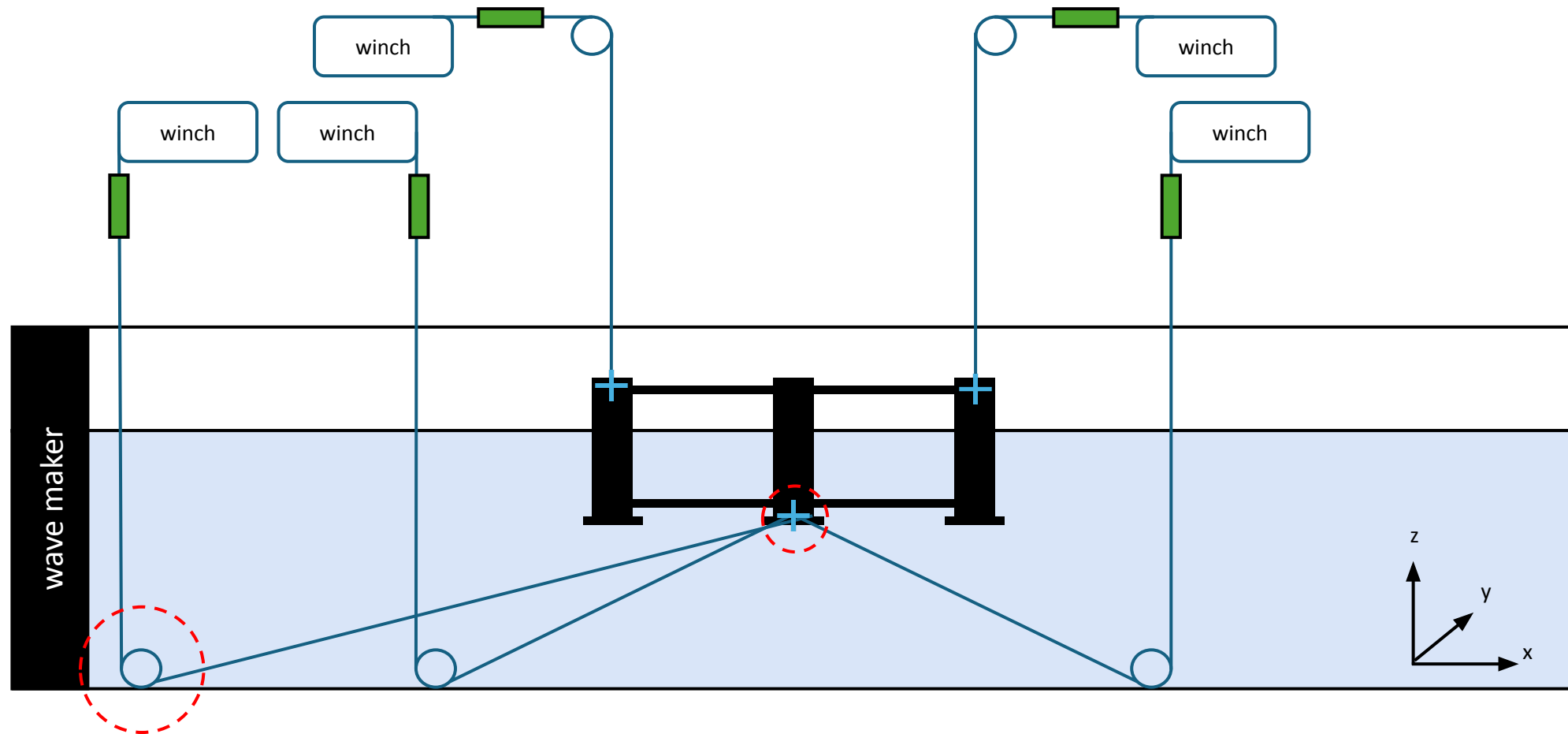
$$\boldsymbol{\tau} = \begin{Bmatrix} T_1 \\ T_2 \\ T_3 \\ T_4 \\ T_5 \end{Bmatrix}$$

where

$T_n$  is line tension in cables 1-5

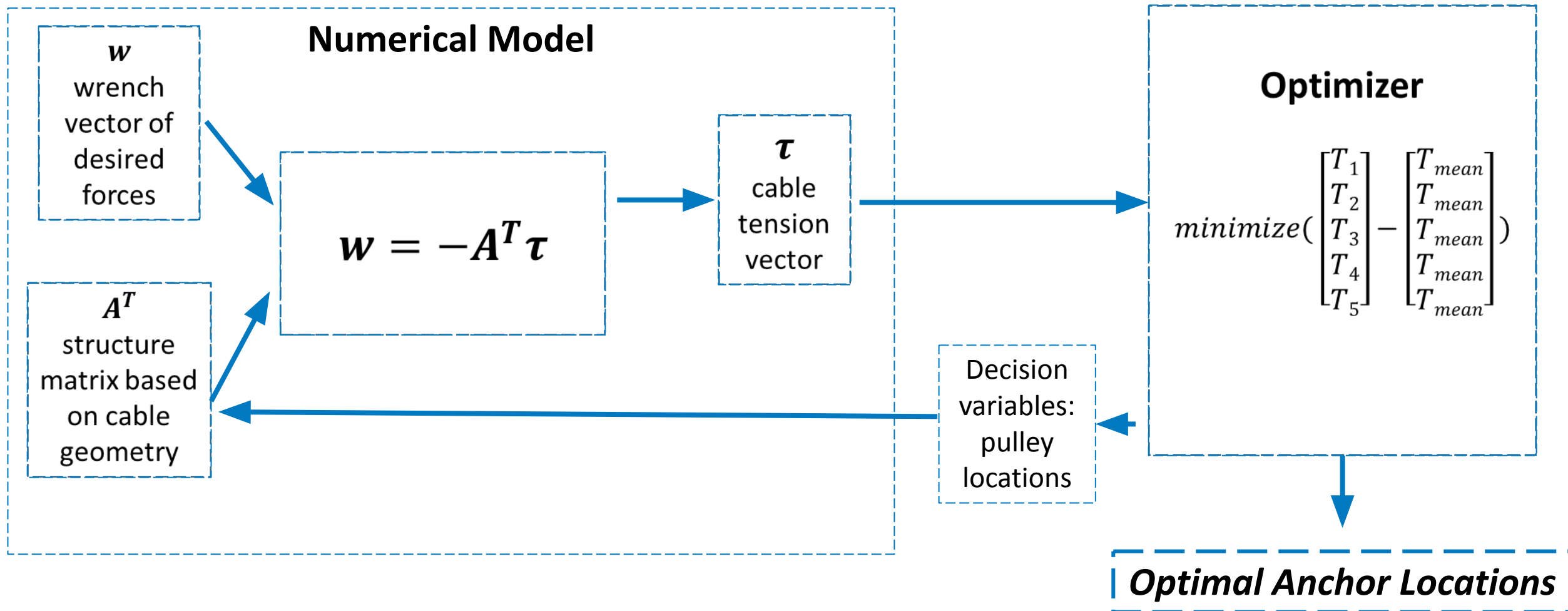
# Cable Geometry Optimization

**Objective:** Find the optimal cable anchor points such that the cable tensions are as close to a mean reference tension value as possible



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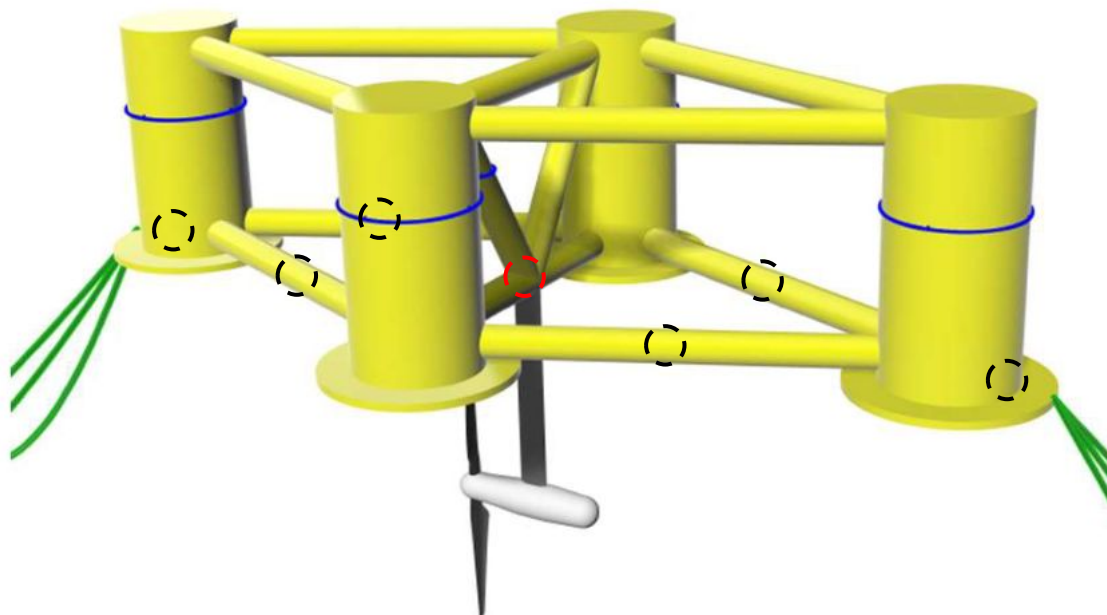


# Cable Attachment Points on Platform

## Considerations

- Structural integrity of chosen attachment points
- Ease of access to limit crossing of cables and platform structure
- Efficient force actuation in the desired DOFs

### Lower Attachment Point Candidates



### Upper Attachment Point Candidates

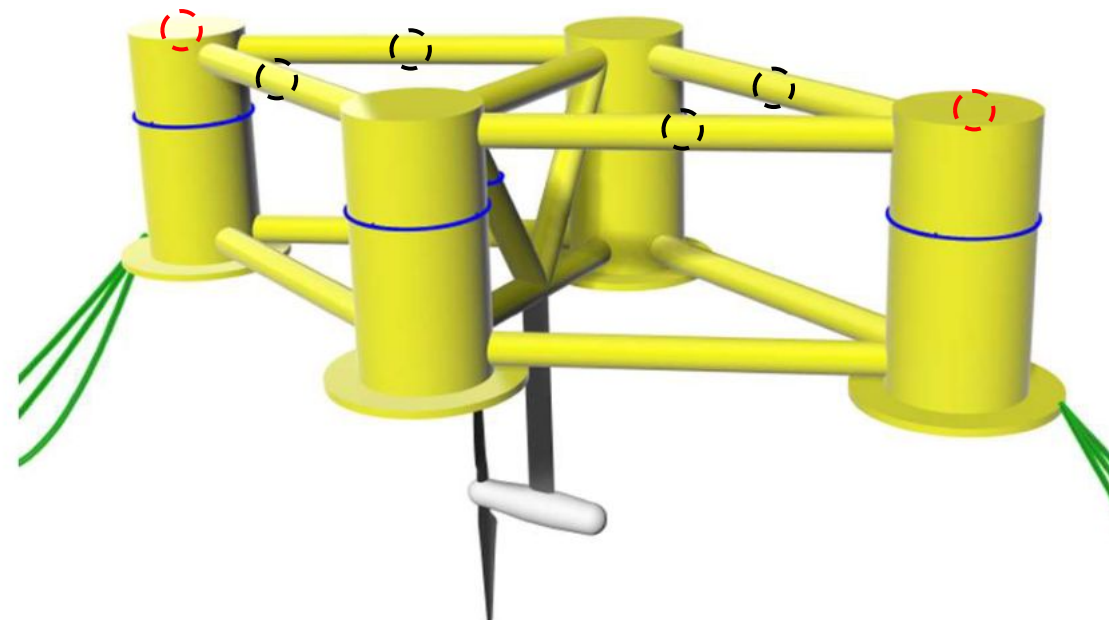
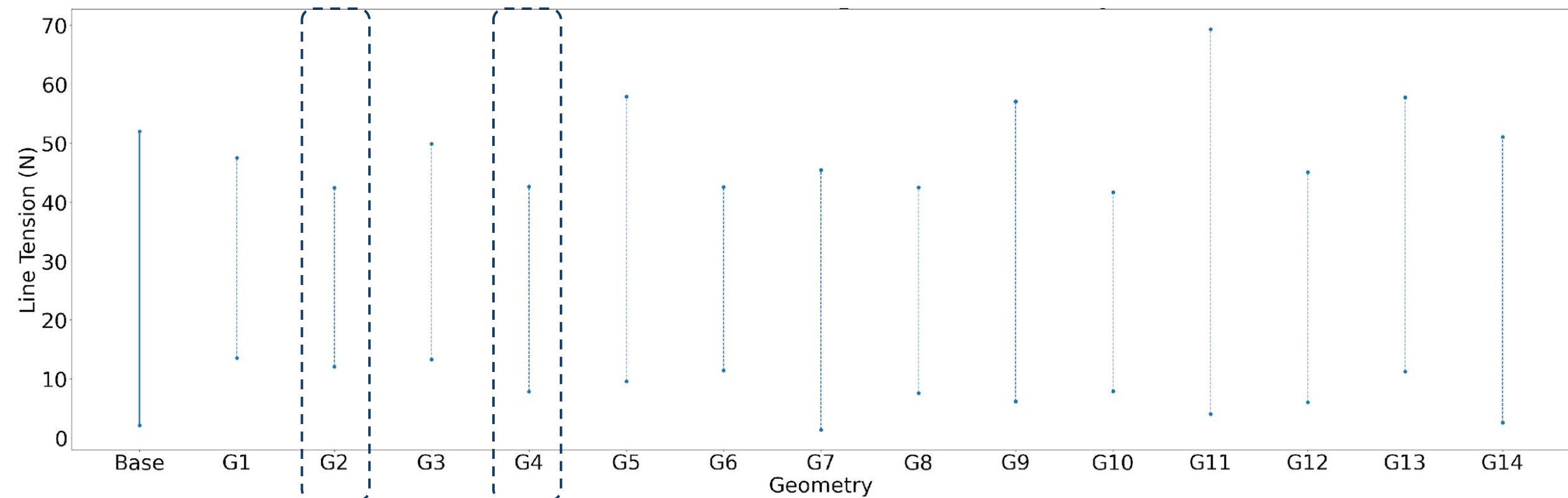


Fig 7. Possible cable attachment points on floating tidal energy convertor RM1

# Optimization Results

## Tension Ranges



*Fig 8. Overall tension range for base geometry and each optimized geometry*

# Optimization Results

## Geometries

### Base Geometry

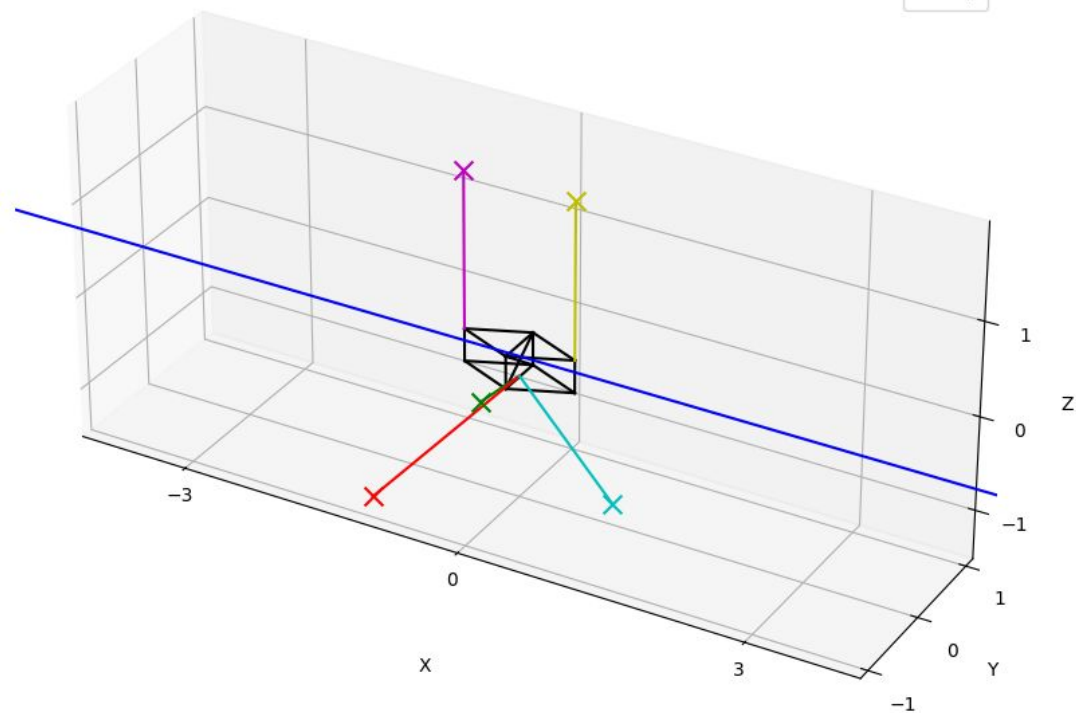


Fig 9. Base Geometry

### Optimized Geometries

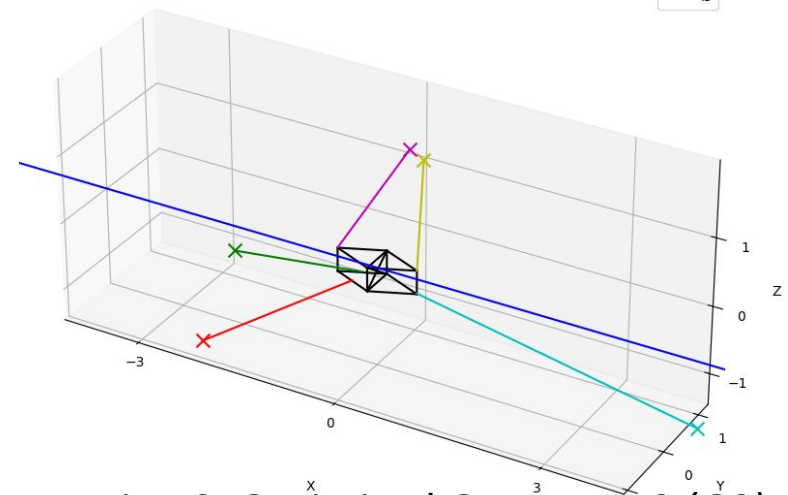


Fig 10. Optimized Geometry 2 (G2)

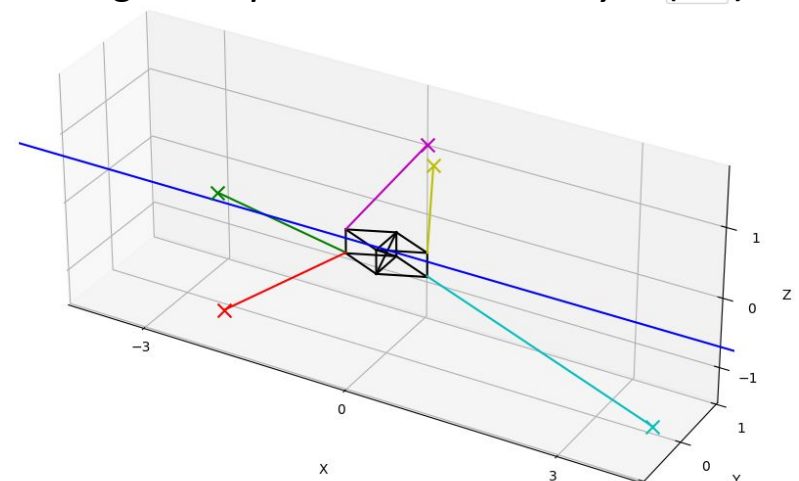
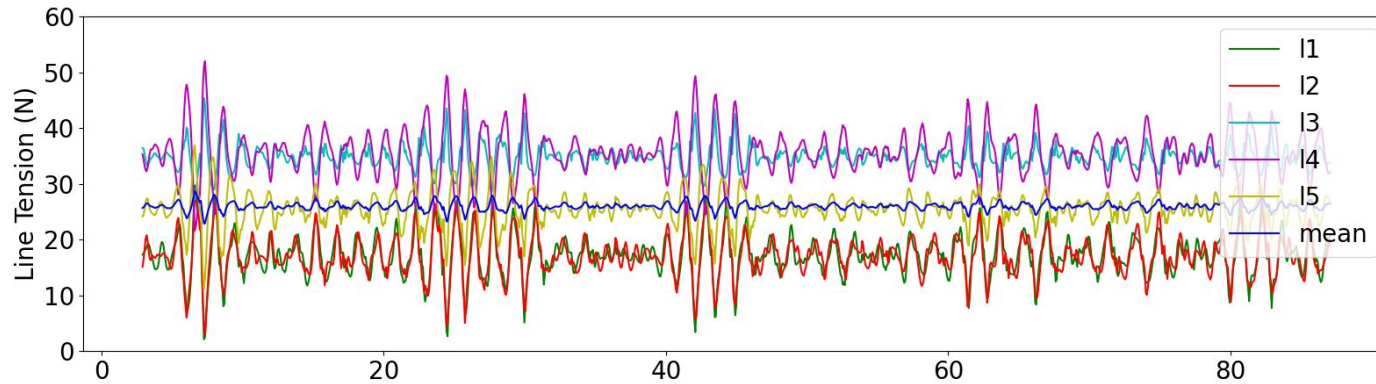


Fig 11. Optimized Geometry 4 (G4)

# Optimization Results

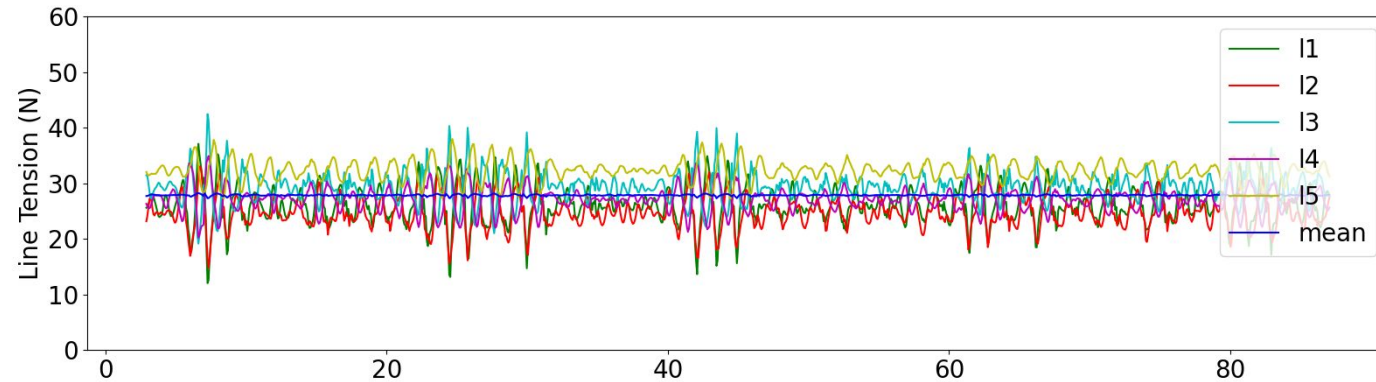
## Tension Time Series

Base



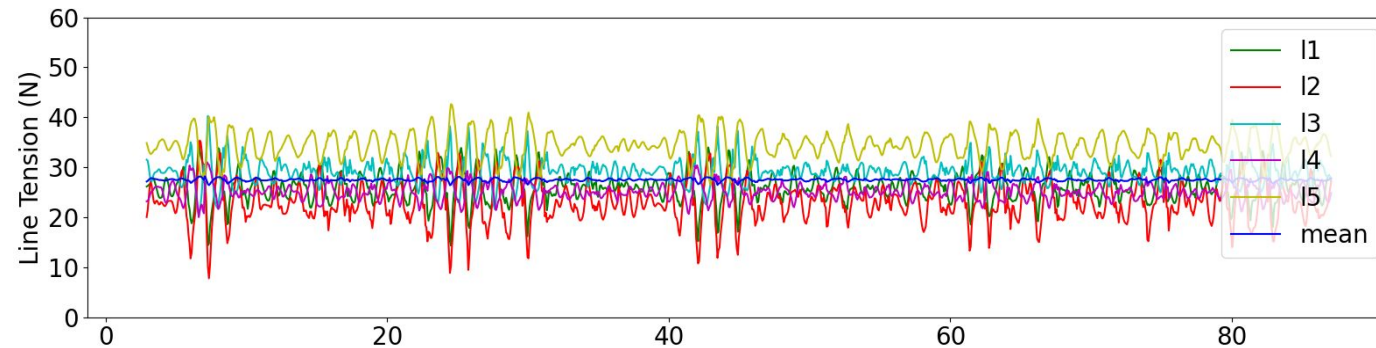
Upper Line Tension Range (N): 40.8 N  
Lower Line Tension Range (N): 43.3 N

G2



Upper Line Tension Range (N): 17.7 N  
Lower Line Tension Range (N): 30.4 N

G4



Upper Line Tension Range (N): 22.6 N  
Lower Line Tension Range (N): 32.5 N

Fig 12. Tension time series for three geometries



# Optimization Conclusion

## Takeaways

Identifying patterns and key features across multiple optimized geometries

Considering the tradeoff of added complexity vs improvement of tension allocation

## Implications

Eliminate the need for wave and wind generation capabilities in order to test floating structures

Testing of other offshore structures (i.e. floating wind turbines and wave energy convertors)

Model more complex flows in both wind and tidal scenarios

# Conclusion

## Future Work

Build Physical  
Actuation System and  
Supporting Structures  
*Summer 2024*

Deploy and Begin Initial  
Wave Basin Testing  
*Fall 2024*

Build Physical RM1  
Model  
*Summer 2024*

# References

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[anisha.sharma@nrel.gov](mailto:anisha.sharma@nrel.gov)

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